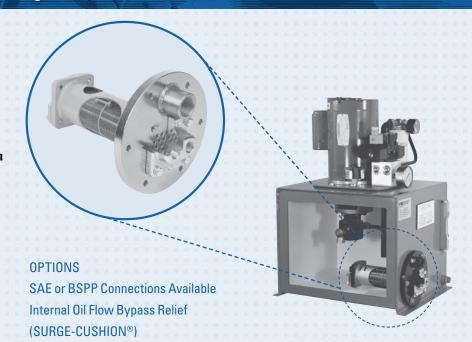
# FLUID COOLING | Shell & Tube EKT Series

#### **COPPER & STEEL CONSTRUCTION**

## **Features**

- HPU, In-tank Cooler
- Compact Size
- EK Style & Size
- High Efficiency Finned Bundle Design
- Serviceable
- Removable
- In-tank Design Minimizes Space Requirements and Reduces Plumbing
- Internal Aluminum Fins Dramatically Increase Performance
- Removable End Bonnets Allow Water Passage Servicing
- High Strength Steel Shell



# Ratings

## **Operating Pressure:**

Shellside 75 psi – Tubeside 150 psi

**Test Pressure:** 

**Shellside** 75 psi – **Tubeside** 150 psi

 $\textbf{Maximum Temperature}~250^{\circ}~\text{F}$ 

# Materials

**Shell** Steel

**Tubes** Copper

Fins Aluminum

**Tubesheets** Steel

**Baffles** Steel

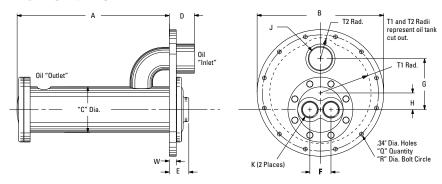
**End Bonnets** Cast Iron

Gaskets Nitrile Rubber/Cellulose Fiber

# Surge-Cushion (Option)

The SURGE-CUSHION® is a protective device (patented) designed to internally bypass a portion of the oil flow during cold start conditions, or when sudden flow surges temporarily exceed the maximum flow allowed for a given cooler. This device may replace an external bypass valve, but it is not intended to bypass the total oil flow.

# **Dimensions**



MODEL	A	В	C	D	Е	F	G	Н	NPT or BSPF	SAE	NPT or BSPF	Q	R	T1	T2	W	Net. Wt.	Approx. Ship Wt.
EKT-508	8.87	6.79	2.55	1.84	1.68	1.12	2.44	.50	3/4"	#12	3/8"	6	5.60	2.25	.79	.62	11	14
EKT-518	18.87	0.79															14	16
EKT-708	8.72	9.75	3.52	- 2.22	1.67	1.62	3.94	1.25	1-1/2"	#24	3/4"	12	8.94	4.00	_	.70	23	27
EKT-718	18.72																30	34
EKT-1012	12.55	10.38	5.05		2.23	2.38	4.69				1"		9.62	4.38	1.12		42	46
EKT-1024	24.55				2.23	2.38	4.09	1.19									58	63

NOTE: We reserve the right to make reasonable design changes without notice. Certified drawings are available upon request. All dimensions in inches. Tank gasket is included. BSPP threads are 55° full form whitworth.

Performance Curves are based on a  $40^{\circ}$ F approach temperature, a 2:1 oil to water ratio and an average oil viscosity of 100 SSU. Example: oil leaving cooler at  $125^{\circ}$ F with  $85^{\circ}$ F cooling water ( $125^{\circ}$ F -  $85^{\circ}$ F =  $40^{\circ}$ F). The 2:1 oil to water ratio means that for every GPM of oil circulated, a minimum of 1/2 GPM of water must must be circulated to obtain the curve results.

#### Step 1 Corrections for approach temperature and oil viscosity.

HPHeat Removed in Cooler =

$$HP_{Actual} \ x \left[ \frac{40^{\circ}F}{0 \text{il out and } ^{\circ}F - Water in }^{\circ}F \right] x \ Correction \ A$$

Step 2

Oil Pressure Drop Coding: ● = 5 PSI; ■ = 10 PSI. Curves havingnopressuredropsymbolindicate that the oil pressuredrop is less than 5 PSI to the highest oil flow rate for that curve. Multiply curve oil pressure drop by Correction B.

# **Viscosity Corrections**

Average Oil SSU	A	В
50	0.84	0.6
100	1.0	1.0
200	1.14	2.0
300	1.24	3.1
400	1.31	4.1
500	1.37	5.1

## **Maximum Flow Rates**

Unit Size	Shell Side GPM)	Tube Side(GPM)
500	20	6
700	60	12
1000	80	28

If maximum allowable flow rates are exceeded, premature failure may occur.

# **Performance Curves**

